

STUDENT VERSION

FORCED VIBRATION AND NO DAMPING

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STATEMENT

Consider a spring mass dashpot model with mass m kg, resistance c N/(m/s), and spring constant k N/m with a forcing function $f(t)$ in units N to describe the displacement from static equilibrium, $y(t)$, (1).

$$m \cdot y''(t) + c \cdot y'(t) + k \cdot y(t) = f(t), \quad y(0) = y_0 \quad \text{and} \quad y'(0) = v_0. \quad (1)$$

In this case we shall deal with an ideal spring and set $c = 0$. Thus we have the following differential equation as (1) becomes (2).

$$m \cdot y''(t) + k \cdot y(t) = F(t), \quad y(0) = y_0 \quad \text{and} \quad y'(0) = v_0. \quad (2)$$

Recall that (2) without a driving function $f(t)$ produces a pure oscillating function with natural frequency $\omega_0 = \sqrt{\frac{k}{m}}$. If now we apply a driver function and our driver is a sinusoidal function, $F(t) = F_0 \cos(\omega t)$, we seek to know what the possibilities are for solutions/outputs from (2).

For notational convenience let us divide both sides of (2) by m and noting that $\omega_0 = \sqrt{\frac{k}{m}}$ we can see that the coefficient of $y(t)$ is $\frac{k}{m} = \left(\sqrt{\frac{k}{m}}\right)^2 = \omega_0^2$. Thus (2) now looks like (3). Also let us set $y(0) = y_0 = 0$ and $y'(0) = v_0 = 0$ as the driver will move the mass.

$$y''(t) + \omega_0^2 y(t) = \frac{F_0}{m} \cos(\omega t), \quad y(0) = 0 \quad \text{and} \quad y'(0) = 0. \quad (3)$$

We can solve (3) by hand or, say, use *Mathematica's* `DSolve` to solve, obtaining:

$$y(t) = \frac{F_0}{m(\omega_0^2 - \omega^2)} (\cos(\omega t) - \cos(\omega_0 t)). \quad (4)$$

We can use the trigonometric identity in (5) to alter our solution (4) to a more learning friendly format.

$$(\cos(\omega t) - \cos(\omega_0 t)) = 2 \sin\left(\frac{\omega_0 + \omega}{2}t\right) \sin\left(\frac{\omega_0 - \omega}{2}t\right). \quad (5)$$

Thus our solution in (3) looks like (6),

$$y(t) = \frac{2F_0}{m(\omega_0^2 - \omega^2)} \sin\left(\frac{\omega_0 + \omega}{2}t\right) \sin\left(\frac{\omega_0 - \omega}{2}t\right). \quad (6)$$

So what might this solution for (3) look like? Let us examine the two time dependent terms, $\text{Term}_1 = \sin\left(\frac{\omega_0 + \omega}{2}t\right)$ and $\text{Term}_2 = \sin\left(\frac{\omega_0 - \omega}{2}t\right)$, in the solution. Assuming both ω and ω_0 are positive numbers, then Term_1 has a higher frequency than Term_2 . We note that Term_1 has a high frequency $\left(\frac{\omega_0 + \omega}{2 \cdot 2\pi}\right)$ Hz while Term_2 has a low frequency $\left(\frac{\omega_0 - \omega}{2 \cdot 2\pi}\right)$ Hz. We say that frequency $\left(\frac{\omega_0 + \omega}{2 \cdot 2\pi}\right)$ is *modulated at* $\left(\frac{\omega_0 - \omega}{2 \cdot 2\pi}\right)$, noting that the low frequency term forms an envelope for the higher frequency term. We demonstrate this in Figure 1(d).

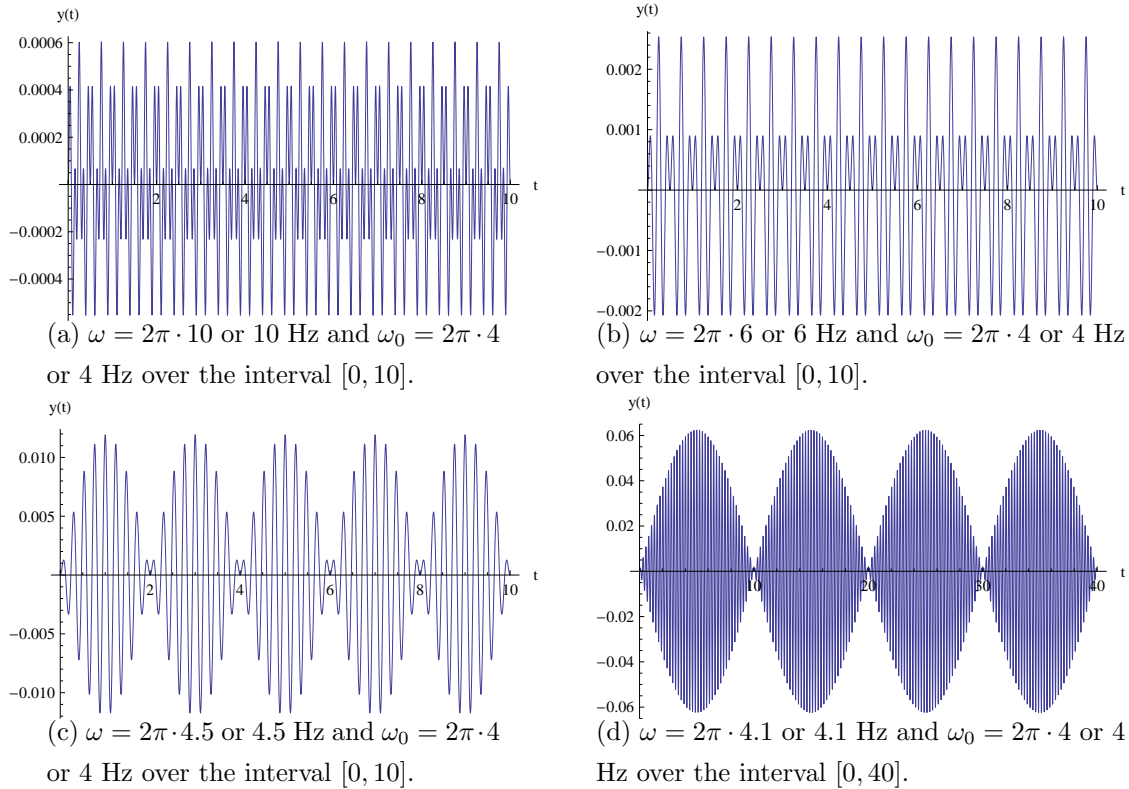


Figure 1. Plots of $y(t) = \frac{2F_0}{m(\omega_0^2 - \omega^2)} \sin\left(\frac{\omega_0 + \omega}{2}t\right) \sin\left(\frac{\omega_0 - \omega}{2}t\right)$, the solution for differential Equation (3), for various values of ω and ω_0 .

In Figure 1 we note that as the driver frequency, ω , gets closer to the natural frequency, ω_0 , that

the envelope frequency of Term₂, i.e. $\left(\frac{\omega_0 - \omega}{2}\right)$ becomes more pronounced and sinusoidal, while the high frequency term of Term₁, i.e. $\left(\frac{\omega_0 + \omega}{2}\right)$ gets ever higher in frequency. When the driver frequency, ω , gets close to the natural frequency ω_0 we say the system displays *beats*, a sort of undulation in amplitude (Term₂) of a high frequency tone (Term₁).

- a) We can play the sound of beats in *Mathematica*. If the frequencies in our system are in the interval [20, 20000] Hz, then humans can detect the corresponding sound. Consider (3) with $\omega_0 = 2\pi \cdot 440$ (440 Hz - the tone of the A note above “middle” C note on the piano keyboard) and $\omega = 2\pi \cdot 480$ (480 Hz). Using `DSolve` in *Mathematica*, solve (3). Then, instead of plotting the obtained solution, use *Mathematica*’s `Play` command, which has the same syntax as `Plot`, to play the sound of our function solution to (3). Here the time interval offered, $\{t, t_{\text{start}}, t_{\text{stop}}\}$, refers to the playing time of the sound. Be sure your sound is on, but not too loud and not too long (say, $t_{\text{start}} = 0$ s and $t_{\text{stop}} = 2$ s), lest you wake a slumbering roommate who packed it in earlier, or a classmate sitting next to you, or the librarian in the reference room where you do your homework! You should recognize this note as the ring tone [?] in the telephone.
- b) Now you try various combinations of ω_0 and ω (close to each other) to see what beats you can produce. Certain combinations evoke the vibrato or tremolo on a “weeping” or sad organ background as the heroine is distressed in “olde timey” films. Have you ever heard the annoying signal on the radio generated by $\omega_0 = 2\pi \cdot 2000$ and $\omega = 2\pi \cdot 2200$? What is this signal used for?
- c) We massage the solution (6) to (3) one more time by dividing and multiplying by $\left(\frac{\omega_0 - \omega}{2}\right)$ to obtain (7):

$$y(t) = \frac{2F_0}{m(\omega_0 + \omega)} \sin\left(\frac{\omega_0 + \omega}{2}t\right) \frac{\sin\left(\frac{\omega_0 - \omega}{2}t\right)}{\frac{\omega_0 - \omega}{2}}. \quad (7)$$

Now we were interested in what happened as our driver frequency, ω , got close to our natural frequency, ω_0 and we saw (and heard!) beats in our solution. Well, what happens as ω gets very close, arbitrarily close, ω_0 . Indeed, what happens as $\omega \rightarrow \omega_0$? In this case the term

$$\frac{\sin\left(\frac{\omega_0 - \omega}{2}t\right)}{\frac{\omega_0 - \omega}{2}}$$

in (7) approaches t . We can obtain this using L’Hôpital’s Rule or using a general `Limit` command in *Mathematica* to see that `Limit[Sin[a t]/a, a -> 0] = t`. Thus our solution (7) to the (3) becomes in the limit (8) as $\omega \rightarrow \omega_0$.

$$y(t) = \frac{F_0}{m \cdot \omega_0} t \sin(\omega_0 t). \quad (8)$$

Unlike the situation which brought about beats, namely a low frequency envelop modulating a high frequency function, now we have an oscillation with a frequency ω_0 – that of the system and the driver, since they are the same, but here the solution is multiplied by t . Thus, as

time increases this solution gets bigger and bigger, wildly oscillating without end. We call this *resonance* and for the most part engineers and scientists are not happy when this happens.

- d) Now you try various combinations of ω_0 and ω to see what resonance you can produce.
- e) Drive (3) (use $m = 1$ and $F_0 = 1$) with a driver equal to the natural frequency, say $\omega = \omega_0 = 2\pi \cdot 440$. Solve the differential equation, plot the solution over $[0, 0.1]$ s and then $[0, 4]$ s. What do you see? Explain what is happening. Now plot/play the solution in the interval $[0, 8]$ s, BUT if you play the solution be sure you are away from those you would not want to disturb!!! Why?
- f) Given (2) or its altered form (4) suppose we had a driver of the form

$$F(t) = F_0 \cos(\alpha t) + G_0 \cos(\beta t), \quad (9)$$

but did not know what α or β actually were. Describe how you could change w_0 so you could “see” a solution which would tell you, first what α was and then what β was. You may assume it is easy, perhaps as easy as a radio dial, to alter ω_0 . Indeed, engineers design and study a simple electrical circuit with resistors, inductors, and capacitors in which the current going through the circuit is modeled as a differential equation and changing the dial (the capacitor) picks out the signals from the ether, much as your changing ω_0 will help you determine α and β . In the latter case we call it tuning your radio dial!